

Electromagnetohydrodynamic effects with single-walled carbon nanotubes particles in a corrugated microchannel

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ABSTRACT

Single-walled carbon nanotubes (SWCNTs) are an advanced product of nanotechnology with notable mechanical and physical properties. This motivated us to investigate the effect of electromagnetic hydrodynamic (EMHD) flow on SWCNTs suspended in a microchannel with corrugated walls. The corrugation of the wavy walls is described by periodic sinusoidal waves of small amplitudes (ϵ), either in phase or out of phase. The problem simulated with a system of governing equations, such as potential, momentum, and heat equations, which were solved analytically using the perturbation method. The behavior of nanofluid velocity, temperature, volumetric flow rate, and average velocity was investigated using three models of thermal characteristics. The results confirm that, the addition of SWCNTs, reduces the fluid velocity at the center of the channel by providing resistance to the fluid motion. The concentration (ϕ) of SWCNTs influence enhances the rate of heat transfer. Additionally, Xue's model has the highest heat transfer rate compared to Maxwell and Hamilton Crosser's (H-C) models. Finally, the obtained flow rate results were compared with previously published data and found to be in good agreement.

1. Introduction

Recently, many researchers were interested in investigating the impact of nanoparticles due to their new physical and thermophysical properties, as well as how these new features affect the industry and biomedical fields. Choi [1] was the first to investigate the efficacy of heat transfer using nanoparticles, and the concept of nanofluid has started to spread. Khanafer et al. [2] investigated the heat transfer enhancement due to the use of carbon nanotubes in the base fluid inside an enclosure, and according to the results, they deduced that the mechanical and thermodynamic properties had improved. Kakac et al. [3] presented a review of how metal nanoparticles in a base fluid, such as water, improved heat transfer. Furthermore, the review demonstrated the investigation of variables based on the development and presentation of the thermophysical properties of a nanofluid. Wahid et al. [4] investigated the impact of velocity slipping and thermal radiation on the velocity and temperature profiles of magnetohydrodynamic hybrid Cu-Al₂O₃/water nanofluid flow over a stretching sheet. Turkylmazoglu [5] investigated the flow and heat transfer of nanofluid over a rotating disc,

taking five different types of nanoparticles into account. They concluded that the nanofluids derived from Cu, CuO, and Ag produce less axial fluid compared with those derived from Al₂O₃ and TiO₂. Moreover, the presence of Cu nanoparticles and a low rate for TiO₂ results in a high heat transfer rate. Elsaid and Abdel-wahed [6] presented a numerical case study for the rate of cooling of a moving cylinder into a coolant supported by two types of nanoparticles (Cu/Al₂O₃), they found that using (Al₂O₃-water) +5 % Cu as a coolant is relatively better than using (Cu-water) +5 % Al₂O₃. Abbasi et al. [7] studied the Brownian motion and the thermophoresis of the non-Newtonian nanofluid runs within a peristaltic channel. Sayed and Abdel-wahed [8] examine the effect of the nanoparticle type and its concentration as well as the type of base fluid, whether Newtonian or non-Newtonian, on the rate of entropy generated and the rate of heat flux from the surface of an MHD nanofluid with a microrotation boundary layer over a moving permeable plate, they found that the entropy generation of the non-Newtonian fluids (fluids with microrotation elements) is higher than that of the Newtonian fluids and that the presence of nanoparticles in the cooling system increases the rate of entropy by 1–4 % according to the type and

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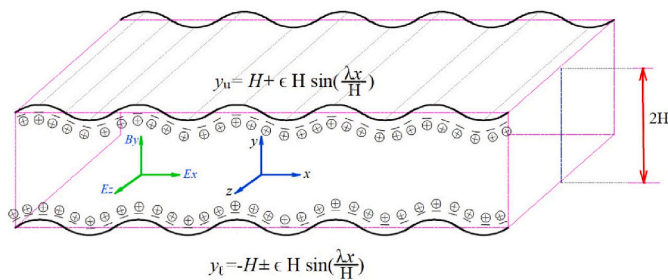


Fig. 1. Geometry of the problem.

concentration of the nanoparticles. Elsaid and Abdel-wahed [9,10] investigated analytically the MHD mixed convection flow of nanofluid inside a vertical channel in the presence/absence of a magnetic induced field. They concluded that an increase in the critical value of the Rayleigh number occur by adding the hybrid nanoparticles within the base fluid. Hayat et al. [11] investigated the mixed convection of peristaltic nanofluid flow runs within a channel, taking two types of nanotubes single and double walls into account. According to the results obtained, adding Single-walled carbon nanotubes (SWCNTs) leads to a decrease in fluid velocity near the center of the channel. Moreover, as the concentration of the SWCNTs increases, the heat transfer rate in the boundary increases. Also, the pressure in the closed section of the pressure leads to a decrease in the pressure rise per wavelength when suspended carbon nanotubes (SWCNTs) are inserted.

Abdel-wahed and Sayed [12] investigated the peristaltic motion of a hybrid/mono-carbon nanotube (SWCNTs)–water flow through a curved channel, they deduce that the hybrid type of nanoparticles raises the pressure gradient at the upper and the lower walls, as well as the heat transfer decreases due to the presence of two types of nanotubes.

Recently, many microfluidic devices have been developed and there have been analytical and numerical models of electromagnetic hydrodynamic (EMHD) flow in microchannels. Chakraborty et al. [13] analyzed the combined effect of magnetic and electric on the fluid flow between two parallel plates. According to the study, a relatively small magnetic field can cause a significant increase in volumetric flow rates. They are, however, highly unreliable in high-strength magnetic fields. Anuar et al. [14] deduced an analytical solution for mixed convection Nano electromagnetohydrodynamic EMHD flow past a vertical sheet, they studied the stability of the deduced solutions for the case of a shrinking sheet. Liu et al. [15] discussed electro-viscous effects on the EMHD flow of Maxwell fluid in a microchannel. They reached an analytical solution and analyzed the flow transport properties concerning the streaming potential and electro-viscous flow. Xue [16] discussed Maxwell’s theorem and the methods for determining effective heat, as well as the thermal conductivity of carbon nanotube-based compounds. Zhao et al. [17] analyzed the incompressible viscous Nano electromagnetohydrodynamic flow runs inside a microchannel under the influence of Lorentz force and joule heating, they investigated the behavior of the velocities and heat transfer under the suggested forces. Duwairi and Abdullah [18] studied analytically and numerically the flow and temperature distribution of a hydrodynamic magnetohydrodynamic fluid under the presence of Lorentz force in a micropump. Yoon et al. [19] discussed the effects of wavy surface form, magnetic field, and heat flux from the disc surface of magnetohydrodynamic (MHD) flow across a fast-rotating axisymmetric wavy disc. They also explored the influence of the wavy surface’s amplitude on the heat transfer properties of MHD flow across a wavy surface. Buren et al. [20] studied the combined effect of magnetic and electric fields on Jeffrey fluids flowing through a microchannel corrugated wall. Andreozzi et al. [21] investigated the effects of heat transport on nanofluid and ribs in a channel. According to the study, triangular ribs have the best thermal performance. Si and Jian [22] studied the effect of both magnetic and

electric fields of the viscous Jeffrey fluid which moves in two parallel corrugated walls under the driven Lorentz force. Rashid et al. [23] investigated the second-grade fluid with a combined effect of magnetic and electric runs between two corrugated walls in a microchannel. Abo-Elkhair et al. [24] examined the use of electric and magnetic fields to control the flow rates of a fluid with variable viscosity moving inside a peristaltic microchannel. Liu et al. [25] investigated the entropy generation rate of the electrohydrodynamic flow of a Newtonian fluid moving through a curved rectangular microchannel. They concluded that the entropy rate increases with the increase of magnetic field parameter Ha until a certain value of it then remain constant as Ha until the entropy rate attains its maximum value, which occurs at the walls. Rashid and Nadeem [26] analytically investigated the flow of a corrugated microchannel wall with porous media under EMHD effects. They used the perturbation technique to analyze the flow of the second-grade fluid, which they regarded as a working fluid. They explored the flow transport phenomenon without describing the temperature distribution associated with such flow. Reza et al. [27] analytically investigated the effect of hydrodynamic electromagnetic flow supported by nanoparticles in a tiny channel with a uniform undulating rough wall in the presence of an applied magnetic field and a transverse electric field.

From the foregoing description and a detailed literature survey, one can easily conclude that no attempt has been made so far in analyzing the MHD flow of carbon nanotubes immersed in water through a corrugated channel. Thermal conductivity has been studied using a variety of models of Carbon nanotubes (SWCNTs), such as Xue’s, Maxwell, and Hamilton-Crosser’s (H-C) models because of their importance in the fields of engineering and medicine.

Carbon nanotubes exhibit various characteristic properties such as high thermal conductivity and the improved thermal performance would be applied to energy systems. The system of the governing equations is affected by the Lorenz force, which is created by the interaction of electric and magnetic fields. The perturbation strategy has been used to address the problem in the micro-channel. And the investigation of the features of fluid flow speed and temperature in a tiny channel with corrugated walls, where the two walls are small-amplitude periodic sinusoidal waves that are either in phase or half-period out of phase.

2. Essential equations as well as the mathematical conceptualization

Consider a laminar, Newtonian, incompressible EMHD flow between two immovable corrugated walls with height $2H$. The origin of the coordinate system is believed to be fixed in the middle of the channel. The layer thickness, $W, L \gg 2H$, is substantially more than the width of the channel along the x -axis and the length along the z -axis (see Fig. 1). The top and lower wavy walls are situated at

$$y_u = H + \epsilon H \sin\left(\frac{\lambda x}{H}\right), \text{ and } y_l = -H \pm \epsilon H \sin\left(\frac{\lambda x}{H}\right), \tag{1}$$

respectively, where λ is the wave number and ϵ is small amplitude,

2.1. Electric potential distribution

To calculate the net charge density ρ_e in the Electric Debye length (EDL) in a corrugated wall channel, first we calculate the EDL potential $\Phi^* = \Phi^*(x, y)$. Then $E = -\nabla\Phi^*$, $\nabla \cdot E = \frac{\rho_e}{\epsilon_0}$ gives Poisson’s equation $\nabla^2\Phi^* = -\frac{\rho_e}{\epsilon_0}$. Thus,

$$\rho_e = -\epsilon_0 \nabla^2 \Phi^* = -2e z_0 n_0 \sinh\left(\frac{e z_0 \Phi^*}{k_B T_a}\right), \tag{2}$$

where n_0, e_0, z_0 , and k_B denote the ion density, electronic charge, valence, and Boltzmann constant, respectively. T_a And ϵ_0 denote the

Table 1
The Thermophysical properties of the base fluid and nanoparticles [10].

| Property | Base fluid/water | SWCNTs |
|---|------------------|--------|
| Density (kg/m ³) | 997.1 | 2200 |
| Thermal conductivity (W/mk) | 0.613 | 2200 |
| Specific heat (j/kgK) | 4179 | 709 |
| Thermal expansion coefficient (1/k)10 ⁻⁶ | 210 | 15 |

Table 2
Comparison of numerical values of volumetric flow rate for different Hartmann number (*Ha*).

| Ha | Buren et al. [20] | | Reza et al. [27] | | Present work | |
|------|-------------------|---------|------------------|---------|--------------|---------|
| | q(1) | q(60) | q(1) | q(60) | q(1) | q(60) |
| 0 | 0.32990 | 0.33300 | 0.33090 | 0.33306 | 0.33327 | 0.33328 |
| 0.01 | 0.32970 | 0.33290 | 0.33010 | 0.33330 | 0.33330 | 0.33331 |
| 0.5 | 0.30304 | 0.30299 | 0.30309 | 0.30300 | 0.30305 | 0.30306 |
| 0.75 | 0.27220 | 0.27223 | 0.27220 | 0.27227 | 0.27223 | 0.27223 |
| 1 | 0.23840 | 0.23846 | 0.23839 | 0.23843 | 0.23839 | 0.23840 |
| 2 | 0.12950 | 0.12950 | 0.12948 | 0.12950 | 0.12949 | 0.12949 |
| 5 | 0.03201 | 0.03205 | 0.03199 | 0.03201 | 0.03200 | 0.03200 |

absolute temperature and the solution’s permittivity constant, respectively. Suppose Φ^* is very small. Then $\left(\frac{ez_0\Phi^*}{k_B T_a}\right) \ll 1$ so the term $\left(\frac{ez_0\Phi^*}{k_B T_a}\right)$ can be used to approximate the term $\left(\sinh\frac{ez_0\Phi^*}{k_B T_a}\right)$ [27,28]. This principle is known as the Debye–Hückle linearization. Finally, the linearized Poisson equation is obtained in the following form.

$$\nabla^2 \Phi^* = \frac{2n_0 e^2 z_0^2}{\epsilon_0 k_B T_a} \Phi^*, \tag{3}$$

where $k = ez_0 \sqrt{\frac{2n_0}{\epsilon_0 k_B T_a}}$ is the Debye–Hückle parameter [27]. Then

$$\nabla^2 \Phi^* = k^2 \Phi^*. \tag{4}$$

And the boundary conditions are

$$\Phi^* = \zeta l \text{ at } y_u = H + \epsilon H \sin\left(\frac{\lambda x^*}{H}\right), y_l = -H \pm \epsilon H \sin\left(\frac{\lambda x^*}{H}\right).$$

Also, the non-dimensional variables are defined as

$$\Phi = \frac{\Phi^*}{\zeta}, y = \frac{y^*}{H}, x = \frac{x^*}{H}.$$

Assume that the ζ potential is constant. To make dimensionless the linearized Poisson equation, we substitute the non-dimensional variables into Eq. (4) to get

$$\nabla^2 \Phi = \alpha^2 \Phi \tag{5}$$

where $\alpha = kH$ is the normalized reciprocal thickness of the EDL, which denotes the ratio of the microchannel’s half height to Debye length $\left(\frac{1}{k}\right)$.

The dimensionless boundary conditions for the electrical potential

Table 3
The volumetric flow rate for different concentration of nanoparticles ϕ on corrugated walls.

| Ha | $\phi = 0.05$ | | $\phi = 0.1$ | | $\phi = 0.2$ | |
|------|---------------|-----------|--------------|-----------|--------------|-----------|
| | q(1) | q(60) | q(1) | q(60) | q(1) | q(60) |
| 0 | 0.2932044 | 0.2932158 | 0.2561344 | 0.2561443 | 0.1908035 | 0.1908109 |
| 0.01 | 0.2931931 | 0.2932045 | 0.2561248 | 0.2561348 | 0.1907972 | 0.1908045 |
| 0.5 | 0.2671459 | 0.2671554 | 0.2340579 | 0.2340662 | 0.1757719 | 0.1757782 |
| 0.75 | 0.2404958 | 0.2405035 | 0.2113437 | 0.2113504 | 0.1600461 | 0.1600513 |
| 1 | 0.2111077 | 0.2111136 | 0.1861343 | 0.1861396 | 0.1422766 | 0.1422807 |
| 2 | 0.1155396 | 0.1155414 | 0.1029685 | 0.1029701 | 0.0811838 | 0.0811851 |
| 5 | 0.0287378 | 0.0287379 | 0.0258518 | 0.0258519 | 0.0209661 | 0.0209661 |

are defined as:

$$\Phi = 1 \text{ at } y = 1 + \epsilon \sin(\lambda x), \tag{6}$$

$$\Phi = 1 \text{ at } y = -1 \pm \epsilon \sin(\lambda x). \tag{7}$$

The electric potential function can be described by the regular perturbation expansion in small values of ϵ [18,27].

$$\Phi(x, y) = \Phi_0(x, y) + \epsilon \Phi_1(x, y) + \epsilon^2 \Phi_2(x, y) + \dots \tag{8}$$

By substituting Eq. (8) into Eq. (5), we get

Table 4
Typical values of the physical variables [26].

| Physical variables | Values [units] |
|---|-------------------------------|
| Characteristic channel length (<i>H</i>) | 40 μm |
| Charge of the photon (<i>e</i>) | 1.6×10^{-19} Coulomb |
| Electrical potential at the wall (ξ) | - 12 Vm |
| Electrical field in axial direction (<i>E_x</i>) | $0 - 2 \times 10^4$ V/m |
| Electrical field in transverse direction (<i>E_y</i>) | $0 - 2 \times 10^4$ V/m |
| Applied magnetic field (<i>B_y</i>) | 1–50 T |
| Boltzmann constant (<i>k_B</i>) | 1.38×10^{-23} J/K |
| Ion density (<i>n₀</i>) | 1 mol/m ³ |
| Average absolute temperature (<i>T_a</i>) | 300 K |
| Initial surface temperature (<i>T_w</i>) | 300 K |
| Valence of ions (<i>z</i>) | 1 |
| Permittivity of the fluid (<i>ε₀</i>) | 5.3×10^{-10} C/Vm |
| Viscosity of the fluid (<i>μ</i>) | $10 - 6$ m ² /s |
| Permeability of porous media (<i>K</i>) | $0 - 10^{-11}$ m ² |
| Electrical conductivity (<i>σ_e</i>) | 1000 W/m ² |

Table 5
Numerical values of the Nusselt number rate at the upper wall for different values of concentration.

| ϕ | $x = 0$ | % of incensement | $x = 60$ | % of incensement |
|-----------------|---------|------------------|----------|------------------|
| Maxwell’s model | | | | |
| 0.05 | 2.08136 | - | 1.24011 | - |
| 0.1 | 2.20113 | 5.75% | 1.36624 | 10.17% |
| 0.15 | 2.42342 | 10.10% | 1.56301 | 14.40% |
| 0.2 | 2.78712 | 15.01% | 1.86028 | 19.02% |
| (H-C) model | | | | |
| 0.05 | 2.01712 | - | 1.19202 | - |
| 0.1 | 2.03471 | 0.87% | 1.24167 | 4.17% |
| 0.15 | 2.09627 | 3.03% | 1.31812 | 6.16% |
| 0.2 | 2.20502 | 5.19% | 1.42455 | 8.07% |
| Xue’s model | | | | |
| 0.05 | 2.63781 | - | 1.65664 | - |
| 0.1 | 3.34464 | 26.80% | 2.22224 | 34.14% |
| 0.15 | 4.17600 | 24.86% | 2.87494 | 29.37% |
| 0.2 | 5.15013 | 23.33% | 3.62915 | 26.23% |

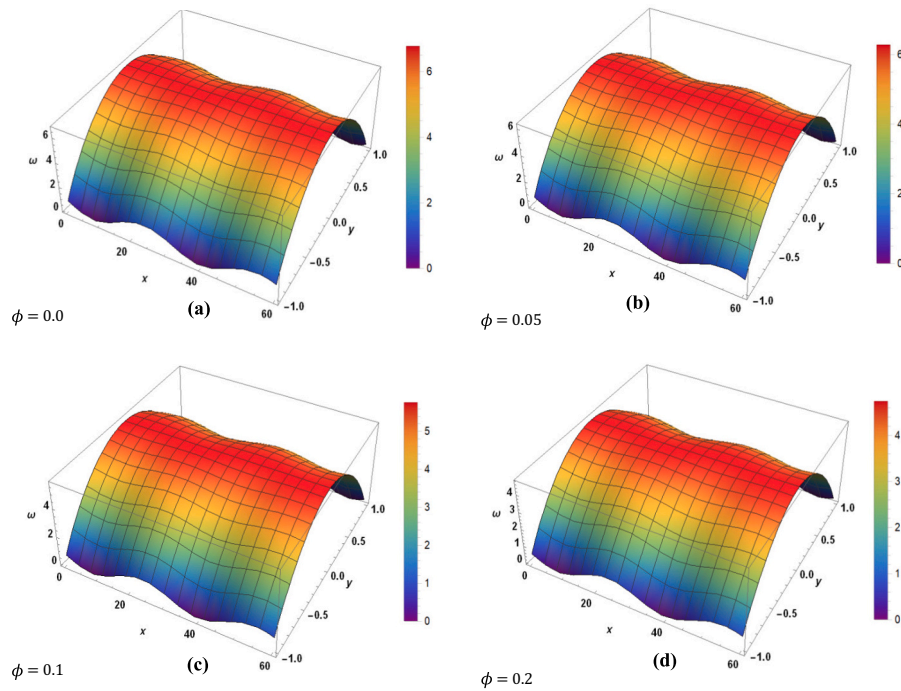


Fig. 2. The velocity distribution in three dimensions for (same phase) $\varepsilon = 0.05, \lambda = 0.2, \alpha = 10, S = 25, Ha = 0.5$.

$$\varepsilon^0 : \frac{\partial^2 \Phi_0}{\partial x^2} + \frac{\partial^2 \Phi_0}{\partial y^2} - \alpha^2 \Phi_0 = 0, \tag{9}$$

$$\varepsilon^1 : \frac{\partial^2 \Phi_1}{\partial x^2} + \frac{\partial^2 \Phi_1}{\partial y^2} - \alpha^2 \Phi_1 = 0, \tag{10}$$

$$\varepsilon^2 : \frac{\partial^2 \Phi_2}{\partial x^2} + \frac{\partial^2 \Phi_2}{\partial y^2} - \alpha^2 \Phi_2 = 0. \tag{11}$$

In a Taylor series concerning the mean wall positions $y = 1$ and $y = -1$, the boundary constraints Eqs. (6) and (7) can be expanded in the following form

$$1 = \Phi(x, 1) + \varepsilon \sin(\lambda x) \frac{\partial \Phi}{\partial y}(x, 1) + \frac{\varepsilon^2}{2} \sin^2(\lambda x) \frac{\partial^2 \Phi}{\partial y^2}(x, 1) + \dots \tag{12}$$

$$1 = \Phi(x, -1) \pm \varepsilon \sin(\lambda x) \frac{\partial \Phi}{\partial y}(x, -1) + \frac{\varepsilon^2}{2} \sin^2(\lambda x) \frac{\partial^2 \Phi}{\partial y^2}(x, -1) + \dots \tag{13}$$

Similarly, the corresponding boundary conditions Eqs. (9), (10), and (11), can be simplified further by collecting terms of equal powers, respectively as follows.

$$\varepsilon^0 : \left\{ \Phi_o(x, y) \Big|_{y=1} = 1, \Phi_o(x, y) \Big|_{y=-1} = 1 \right. \tag{14}$$

$$\varepsilon^1 : \left\{ \begin{aligned} \Phi_1(x, y) \Big|_{y=1} + \sin(\lambda x) \frac{\partial \Phi_o(x, y)}{\partial y} \Big|_{y=1} &= 0, \\ \Phi_1(x, y) \Big|_{y=-1} \pm \sin(\lambda x) \frac{\partial \Phi_o(x, y)}{\partial y} \Big|_{y=-1} &= 0, \end{aligned} \right. \tag{15}$$

$$\varepsilon^2 : \left\{ \begin{aligned} \Phi_2(x, y) \Big|_{y=1} + \sin(\lambda x) \frac{\partial \Phi_1(x, y)}{\partial y} \Big|_{y=1} + \frac{\sin^2(\lambda x)}{2} \frac{\partial^2 \Phi_o(x, y)}{\partial y^2} \Big|_{y=1} &= 0, \\ \Phi_2(x, y) \Big|_{y=-1} \pm \sin(\lambda x) \frac{\partial \Phi_1(x, y)}{\partial y} \Big|_{y=-1} + \frac{\sin^2(\lambda x)}{2} \frac{\partial^2 \Phi_o(x, y)}{\partial y^2} \Big|_{y=-1} &= 0. \end{aligned} \right. \tag{16}$$

From Eqs. (9) and (14), we get

$$\Phi_0^\pm(x, y) = \frac{\cosh(y\alpha)}{\cosh(\alpha)}. \tag{17}$$

From Eqs. (10) and (15), we have

$$\therefore \Phi_1^\pm(x, y) = -\alpha \sin[\lambda x] \operatorname{Tanh}[\alpha] \begin{cases} \frac{\operatorname{Sinh}[yS_1]}{\operatorname{Sinh}[S_1]} & \text{for } \Phi_1^+, \\ \frac{\operatorname{Cosh}[yS_1]}{\operatorname{Cosh}[S_1]} & \text{for } \Phi_1^-. \end{cases} \tag{18}$$

From Eqs. (11) and (16), we get

$$\therefore \Phi_2^\pm(x, y) = \frac{1}{4} \alpha \Omega_1 \begin{cases} \alpha - \frac{2 S_1 \operatorname{Tanh}[\alpha]}{\operatorname{Tanh}[S_1]} & \text{for } \Phi_2^+, \\ \alpha - \frac{2 S_1 \operatorname{Tanh}[\alpha]}{\operatorname{Coth}[S_1]} & \text{for } \Phi_2^-, \end{cases} \tag{19}$$

where

$$\Omega_1 = \frac{\operatorname{Cos}[2\lambda] \operatorname{Cosh}[yS_2]}{\operatorname{Cosh}[S_2]} - \frac{\operatorname{Cosh}[y\alpha]}{\operatorname{Cosh}[\alpha]},$$

where S_1, S_2 are defined in Appendix A.

The above Eqs. (9)–(11) with their boundary conditions (14)–(16) are solved exactly using DSolve function with assistance of Mathematica program. By getting $\Phi_o(x, y), \Phi_1(x, y)$, and $\Phi_2(x, y)$, the general solution for the potential equation can be obtained using Eq. (8).

2.2. Velocity distribution

The dimensional form of the continuity and momentum balance

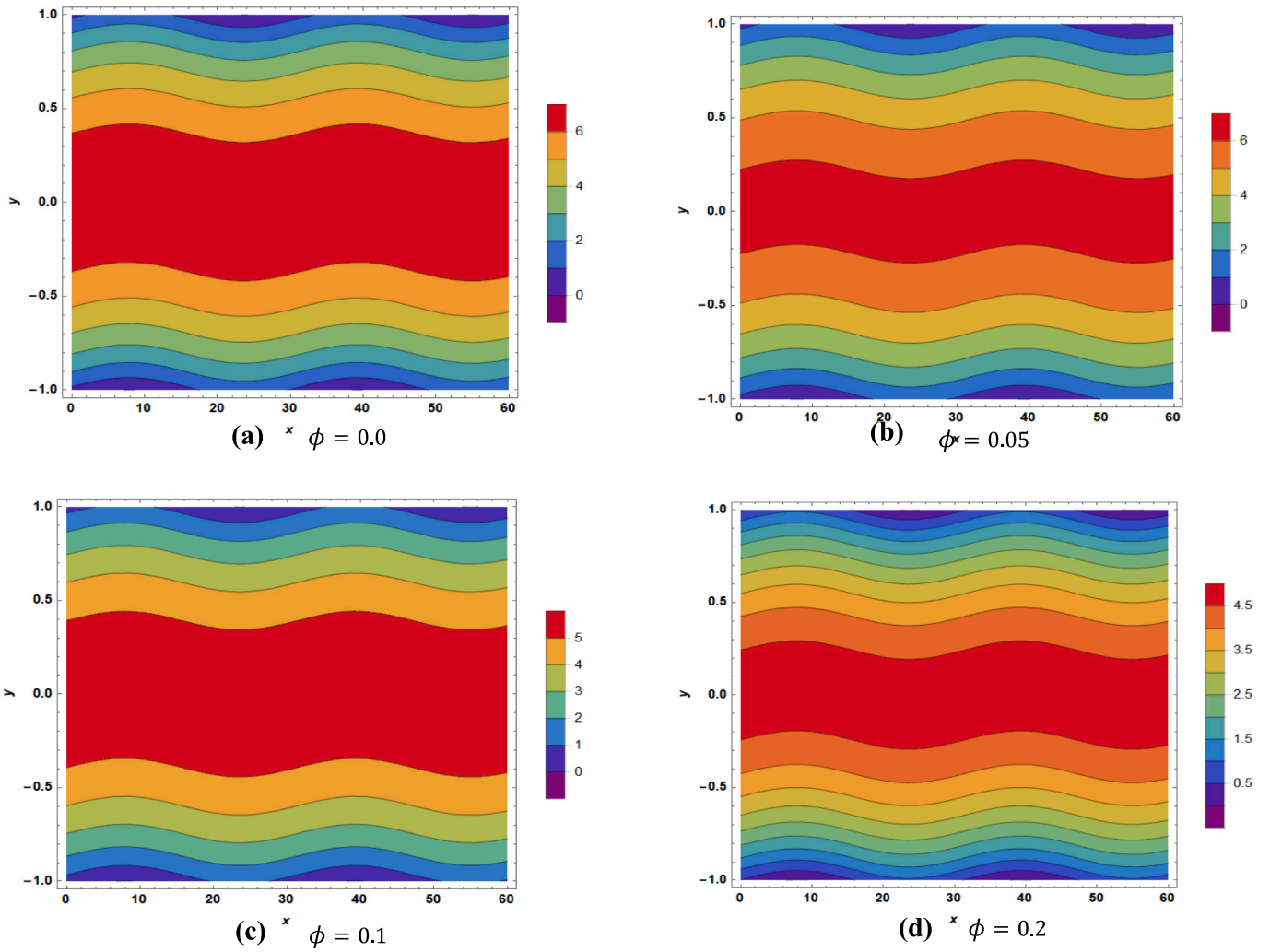


Fig. 3. Contour plot for velocity distribution (same phase) $\varepsilon = 0.05, \lambda = 0.2, \alpha = 10, S = 25, Ha = 0.5$.

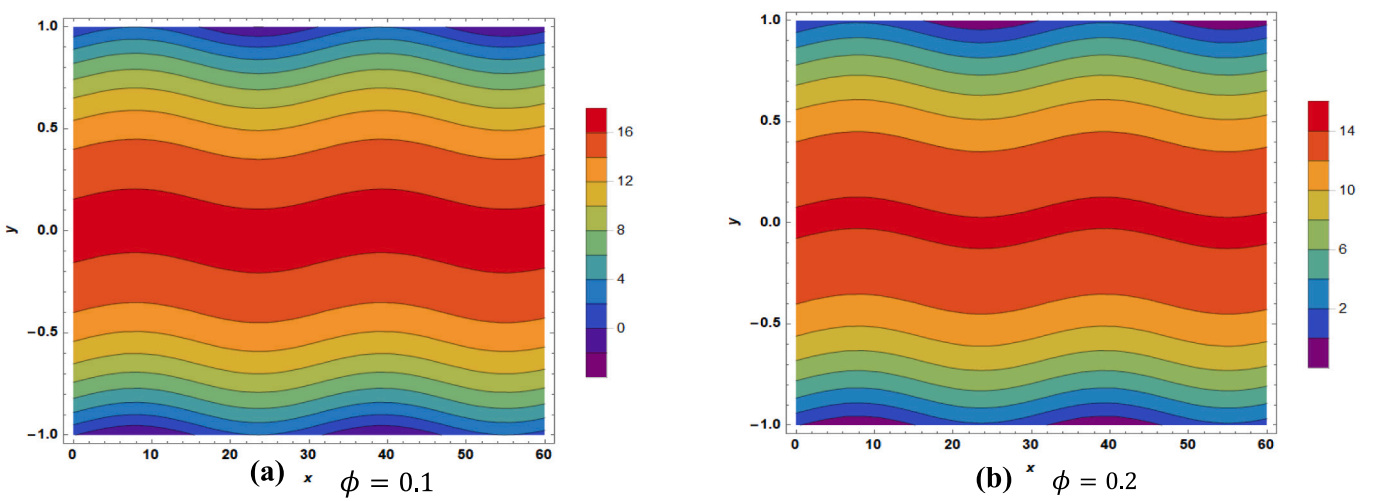


Fig. 4. Contour plot for velocity distribution (same phase) $\varepsilon = 0.05, \lambda = 0.2, \alpha = 10, S = 50, Ha = 1$.

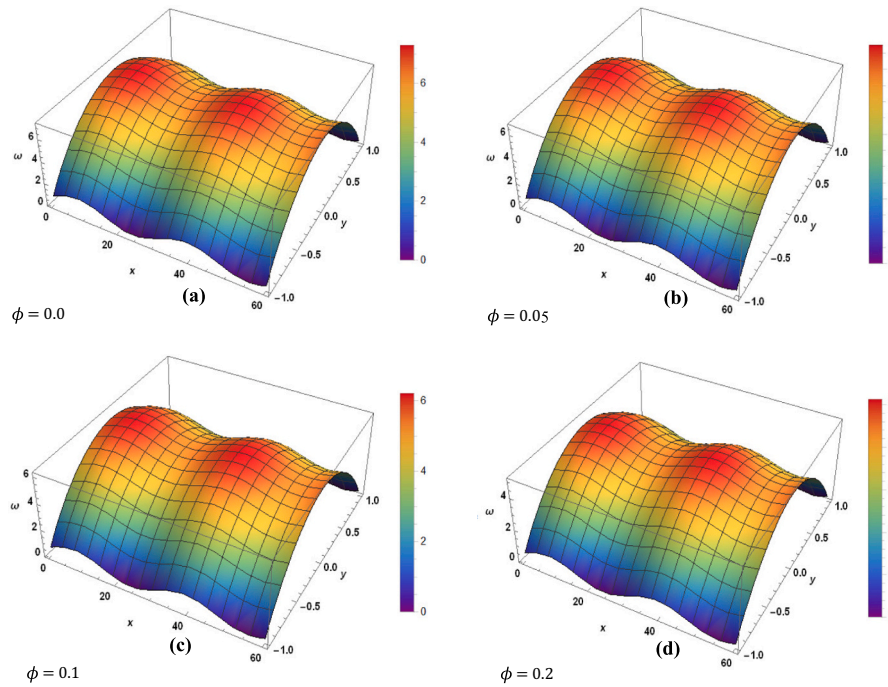


Fig. 5. The velocity distribution in three dimensions for (opposite phase) $\epsilon = 0.05, \lambda = 0.2, \alpha = 10, S = 25, Ha = 0.5$.

equations for a fluid was given by: [25,29].

$$\rho_{nf} \frac{\partial \vec{u}}{\partial t} + \rho_{nf} (\vec{u} \cdot \nabla) \vec{u} = -\nabla \vec{p} + \mu_{nf} \nabla^2 \vec{u} + \vec{F}, \tag{20}$$

where

$$\vec{F} = \vec{J} \times \vec{B} + \rho_e \vec{E}. \tag{21}$$

In Eq. (21) ρ_e and \vec{E} denote the net electric charge density and applied electric field. F denotes the Lorentz force, which is formed by the interaction of fluid flow and applied magnetic field, where the magnetic field acts in y -direction, $\vec{B} = B_0 \vec{e}_y$, and the uniform electric field is $\vec{E} = E_x \vec{e}_x + E_z \vec{e}_z$ and the electric current density $\vec{J} = \sigma_{nf} [\vec{u} \times \vec{B}] + \sigma_{nf} \vec{E}$ caused by an x -directional electric field [26]. Therefore, the momentum equation in z -direction takes the form:

$$\mu_{nf} \left(\frac{\partial^2 w^*}{\partial x^{*2}} + \frac{\partial^2 w^*}{\partial y^{*2}} \right) - \frac{\partial P}{\partial z} - \sigma_{nf} w^* B_0^2 + \sigma_{nf} E_x B_0 + \rho_e E_z = 0. \tag{22}$$

By introducing the following Dimensionless variables, and substituting in Eq. (22),

$$\begin{aligned} x &= \frac{x^*}{H}, y = \frac{y^*}{H}, w = \frac{w^*}{V}, Ha^2 = \frac{B_0^2 H^2 \sigma_f}{\mu_f}, p_z = -\frac{H^2}{\mu_f V} \frac{\partial p}{\partial z}, Ha \bullet S \\ &= B_0 H \sqrt{\frac{\sigma_f}{\mu_f}}, \rho_e = \alpha^2 H^2 \Phi, \end{aligned}$$

where

$$\mu_{nf} = \frac{\mu_f}{(1-\phi)^{2.5}}, \sigma_{nf} = \sigma_f \left[1 + \frac{3\phi \left(\frac{\sigma_x}{\sigma_f} - 1 \right)}{\left(\frac{\sigma_x}{\sigma_f} + 2 \right) - \phi \left(\frac{\sigma_x}{\sigma_f} - 1 \right)} \right],$$

where ϕ the nanoparticles volume fraction, $A_1 = \frac{\mu_{nf}}{\mu_f}, A_2 = \frac{\sigma_{nf}}{\sigma_f}$, as in [11]. The thermo-physical properties of the base fluid and nanoparticles are listed in Table 1

$$\frac{\partial^2 w}{\partial x^2} + \frac{\partial^2 w}{\partial y^2} - \frac{A_2}{A_1} Ha^2 w + \frac{\alpha^2}{A_1} \Phi = -\frac{1}{A_1} P_z - \frac{A_2}{A_1} Ha^* S, \tag{23}$$

where the Hartmann number (Ha) describes the ratio of Lorentz force to viscous force, and S denotes the transverse electric field, with the following boundary conditions:

$$w = 0 \text{ at } y_u = 1 + \epsilon \sin(\lambda x), \tag{24}$$

$$w = 0 \text{ at } y_L = -1 \pm \epsilon \sin(\lambda x). \tag{25}$$

Using the following transformation in Eq. (23), we obtain

$$u = -\frac{A_2}{A_1} Ha^2 w + \frac{\frac{\alpha^2}{A_1} Ha^2}{\left(Ha^2 - \frac{A_1 \alpha^2}{A_2} \right)} \Phi. \tag{26}$$

One can deduce the following form of the velocity equation

$$\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} - \frac{A_2}{A_1} Ha^2 u = Ha^2 \frac{A_2}{A_1^2} P_z + \frac{A_2^2}{A_1^2} Ha^3 S, \tag{27}$$

By defining the following variables

$$A^2 = \frac{A_2}{A_1} Ha^2, Q = \frac{\frac{\alpha^2}{A_1} Ha^2}{\left(Ha^2 - \frac{A_1 \alpha^2}{A_2} \right)},$$

The boundary conditions take the form

$$u = Q \text{ at } y_u = 1 + \epsilon \sin(\lambda x) \tag{28}$$

$$u = Q \text{ at } y_u = -1 \pm \epsilon \sin(\lambda x) \tag{29}$$

The fluid velocity can be described by a regular perturbation expansion in small values of ϵ [25–27].

$$u(x, y) = u_0(x, y) + \epsilon u_1(x, y) + \epsilon^2 u_2(x, y) + \dots \tag{30}$$

The differential equations for powers of ϵ are obtained by substituting Eq. (30) into Eq. (27).

$$\epsilon^0 : \frac{\partial^2 u_0}{\partial x^2} + \frac{\partial^2 u_0}{\partial y^2} - A^2 u_0 = B_1 \tag{31}$$

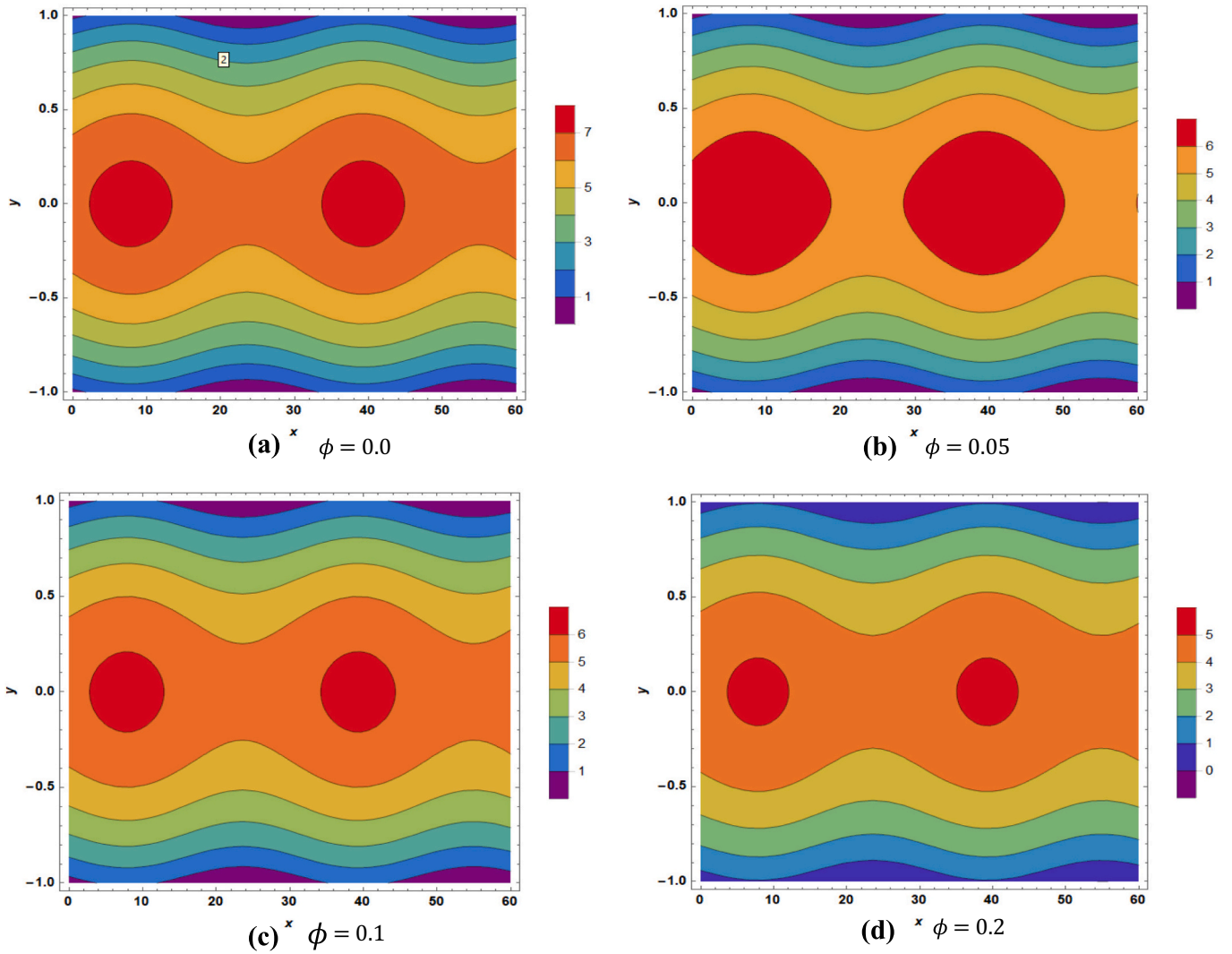


Fig. 6. Contour plot for velocity distribution (opposite phase) $\epsilon = 0.05, \lambda = 0.2, \alpha = 10, S = 25, Ha = 0.5$.

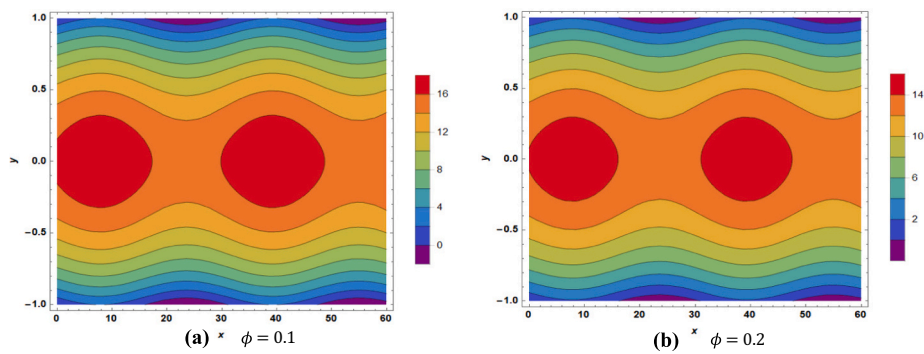


Fig. 7. Contour plot for velocity distribution (opposite phase) $\epsilon = 0.05, \lambda = 0.2, \alpha = 10, S = 50, Ha = 1$.

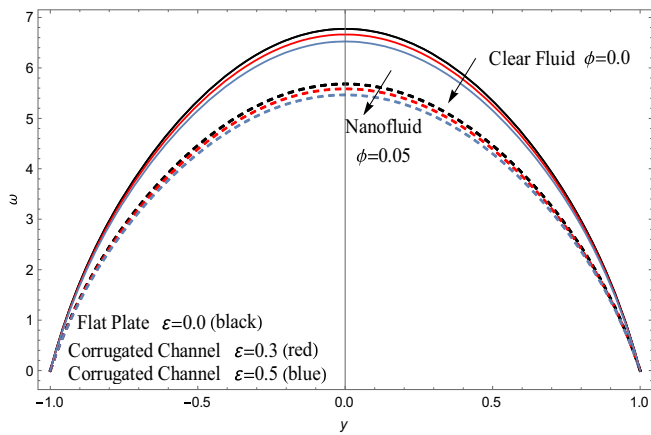


Fig. 8. The velocity distribution for clear/nanofluid at $\epsilon = 0, \epsilon = 0.3, \epsilon = 0.5$.

$$\epsilon^1 : \frac{\partial^2 u_1}{\partial x^2} + \frac{\partial^2 u_1}{\partial y^2} - A^2 u_1 = 0 \tag{32}$$

$$\epsilon^2 : \frac{\partial^2 u_2}{\partial x^2} + \frac{\partial^2 u_2}{\partial y^2} - A^2 u_2 = 0 \tag{33}$$

In a Taylor series concerning the mean wall positions $y = 1$ and $y = -1$, the boundary constraints (28) and (29) can be expanded to be in the form:

$$Q = u(x, 1) + \epsilon \sin(\lambda x) \frac{\partial u}{\partial y}(x, 1) + \frac{\epsilon^2}{2} \sin^2(\lambda x) \frac{\partial^2 u}{\partial y^2}(x, 1) + \dots \tag{34}$$

$$Q = u(x, -1) \pm \epsilon \sin(\lambda x) \frac{\partial u}{\partial y}(x, -1) + \frac{\epsilon^2}{2} \sin^2(\lambda x) \frac{\partial^2 u}{\partial y^2}(x, -1) + \dots \tag{35}$$

We obtain for the corresponding boundary condition Eqs. (31), (32), and (33), respectively, by collecting terms of equal powers of ϵ .

$$\epsilon^0 : \left\{ u_o(x, y) \Big|_{y=1} = Q, u_o(x, y) \Big|_{y=-1} = Q, \right. \tag{36}$$

$$\epsilon^1 : \begin{cases} u_1(x, y) \Big|_{y=1} + \sin(\lambda x) \frac{\partial u_o(x, y)}{\partial y} \Big|_{y=1} = 0, \\ u_1(x, y) \Big|_{y=-1} \pm \sin(\lambda x) \frac{\partial u_o(x, y)}{\partial y} \Big|_{y=-1} = 0, \end{cases} \tag{37}$$

$$\epsilon^2 : \begin{cases} u_2(x, y) \Big|_{y=1} + \sin(\lambda x) \frac{\partial u_1(x, y)}{\partial y} \Big|_{y=1} + \frac{\sin^2(\lambda x)}{2} \frac{\partial^2 u_o(x, y)}{\partial y^2} \Big|_{y=1} = 0, \\ u_2(x, y) \Big|_{y=-1} \pm \sin(\lambda x) \frac{\partial u_1(x, y)}{\partial y} \Big|_{y=-1} + \frac{\sin^2(\lambda x)}{2} \frac{\partial^2 u_o(x, y)}{\partial y^2} \Big|_{y=-1} = 0. \end{cases} \tag{38}$$

From Eqs. (31) and (36), we derived

$$u_0^\pm(x, y) = \frac{e^{-Ay} (A^2 e^A (1 + e^{2Ay}) Q + (e^A - e^{Ay} - e^{A(2+y)} + e^{A+2Ay}) B_1)}{A^2 (1 + e^{2A})}. \tag{39}$$

From Eqs. (32) and (37), we derived

$$u_1^\pm(x, y) = -B_4 \text{Sin}[x\lambda] \begin{cases} \frac{\text{Sinh}[yB_2]}{\text{Sinh}[B_2]} \text{ for } u_1^+, \\ \frac{\text{Cosh}[yB_2]}{\text{Cosh}[B_2]} \text{ for } u_1^-, \end{cases} \tag{40}$$

From Eqs. (33) and (38), we derived

$$u_2^\pm(x, y) = \frac{1}{4} \Omega_2 \begin{cases} \left(A^2 Q + B_1 - \frac{2B_2 B_4}{\text{Tanh}[B_2]} \right) \text{ for } u_2^+, \\ \left(A^2 Q + B_1 - \frac{2B_2 B_4}{\text{Coth}[B_2]} \right) \text{ for } u_2^-, \end{cases} \tag{41}$$

where

$$\Omega_2 = \left(\frac{-\text{Cosh}[Ay]}{\text{Cosh}[A]} + \frac{\text{Cos}[2x\lambda] \text{Cosh}[yB_3]}{\text{Cosh}[B_3]} \right).$$

The volume flow rate per unit channel width can be written, where in the last two integrals the function w is expanded in a Taylor series about the mean wall positions. The result is obtained by averaging over one wavelength of corrugations.

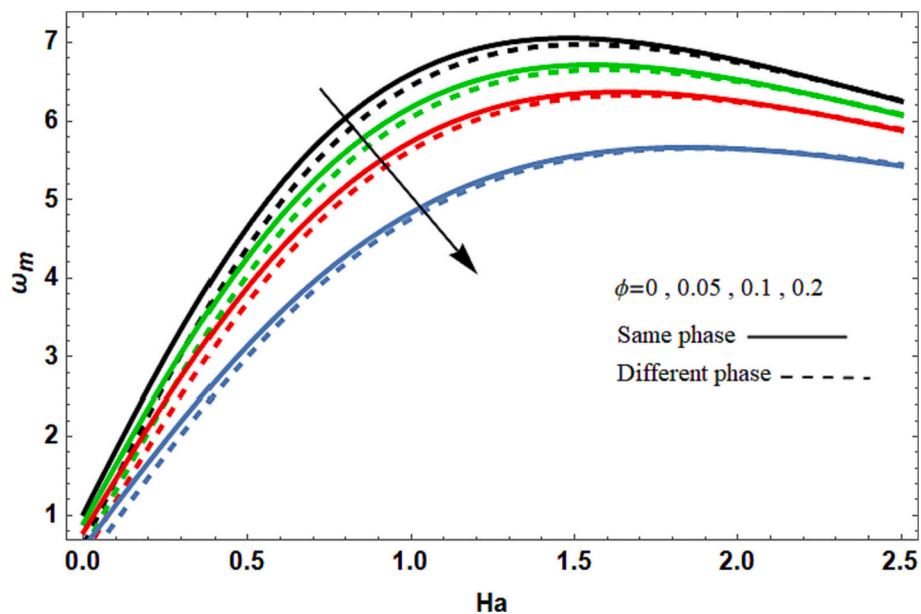


Fig. 9. The Mea velocity w_m and Hartmann number (Ha) for $\epsilon = 0.2, \lambda = 0.2, \alpha = 10, S = 50$.

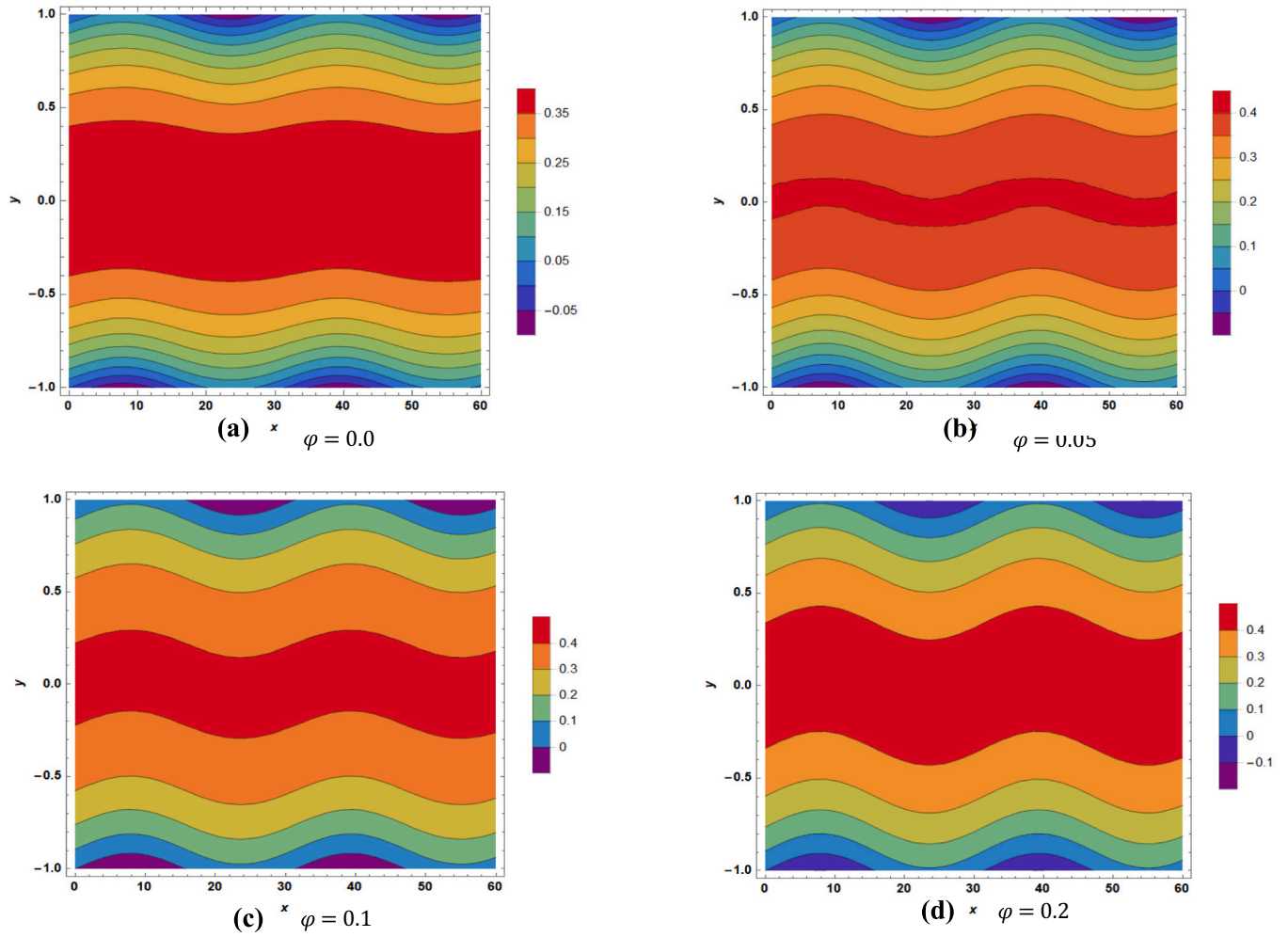


Fig. 10. Contour plot for Temperature distribution (same phase) $\varepsilon = 0.1, \lambda = 0.2, \alpha = 10, S = 25, Ha = 0.5, j = 1, \varepsilon = 1$.

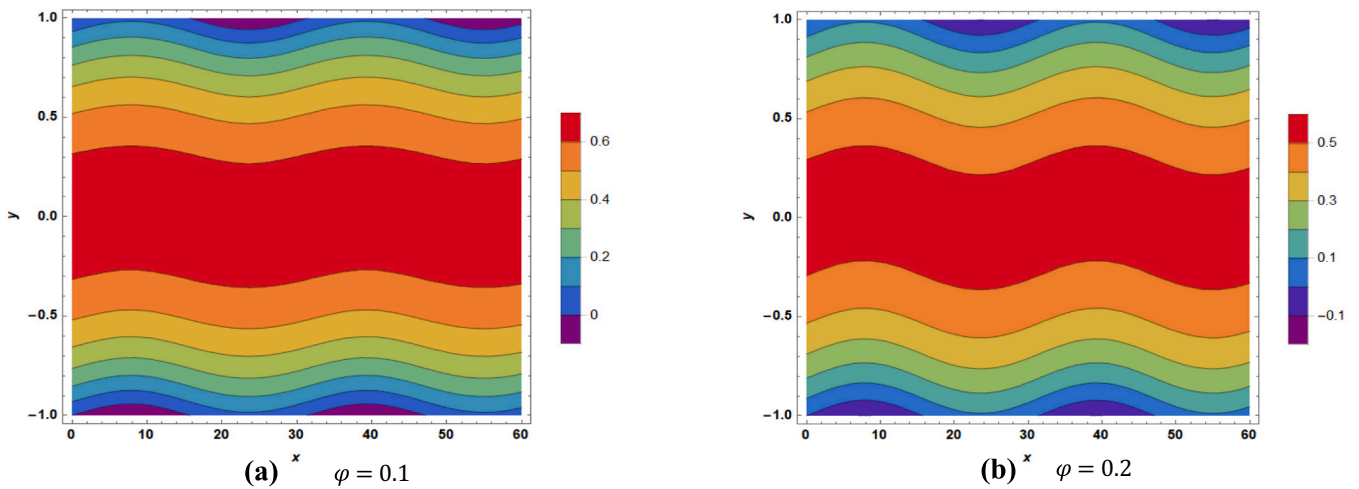


Fig. 11. Contour plot for Temperature distribution (same phase) $\varepsilon = 0.1, \lambda = 0.2, \alpha = 10, S = 50, Ha = 1, j = 1, \varepsilon = 1$.

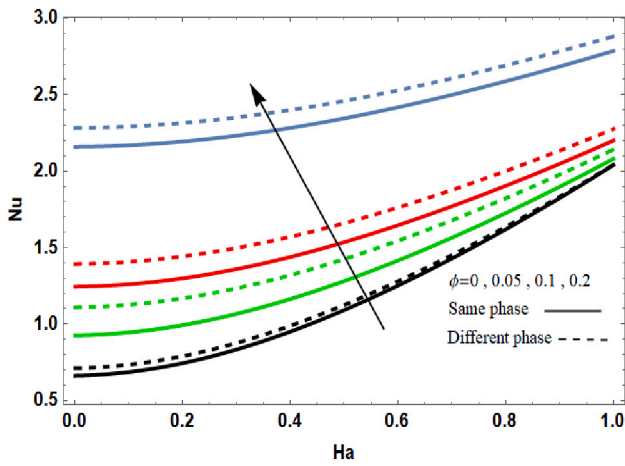


Fig. 12. Variation of Nu with Ha for different values of ϕ at $\varepsilon = 0.1, \lambda = 0.1, \alpha = 10, S = 25$.

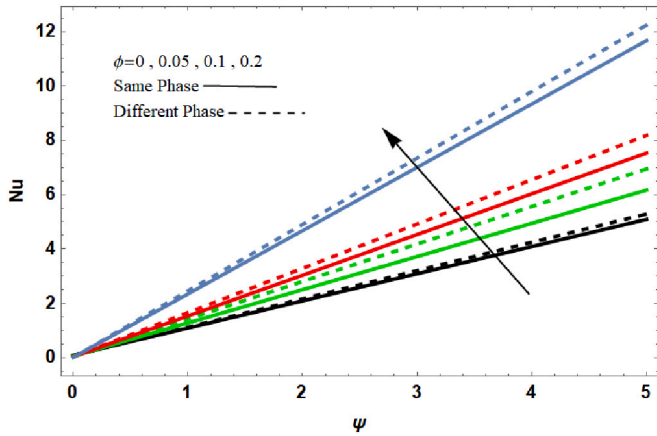


Fig. 13. Variation of Nu with heat generation/absorption ψ for different values of ϕ at $\varepsilon = 0.1, \lambda = 0.1, \alpha = 10, S = 25$.

$$\begin{aligned}
 q^\pm(x) &= \int_{y_1}^{y_u} w \, dy = \int_{-1}^1 w \, dy + \int_1^{y_u} w \, dy + \int_{y_1}^{-1} w \, dy \\
 &= \int_{-1}^1 w_0 \, dy + \varepsilon \left[\int_{-1}^1 w_1^\pm \, dy + \sin[\lambda x](w_0(1) \mp w_0(-1)) \right] + \varepsilon^2 \left[\int_{-1}^1 w_2^\pm \, dy + \sin[\lambda x](w_1(1) \mp w_1(-1)) \right] \\
 &\quad + \frac{\sin^2[\lambda x]}{2} \left(\left. \frac{dw_0}{dx} \right|_{y=1} - \left. \frac{dw_0}{dx} \right|_{y=-1} \right) + \dots \dots \dots
 \end{aligned}
 \tag{42}$$

The mean velocity is given by

$$w_m^\pm = \frac{\lambda}{4\pi} \int_0^{\frac{2\pi}{\lambda}} q^\pm(x) \, dx = w_{0m} [1 + \varepsilon^2 \phi^\pm + O(\varepsilon^4) \dots \dots \dots], \tag{43}$$

$$w_{0m}^\pm = \frac{2(\alpha B_1(A - \text{Tanh}[A]) + A^2 Q(-\alpha \text{Tanh}[A] + A \text{Tanh}[\alpha]))}{A^5 \alpha}, \tag{44}$$

where, $B_1 - B_4$ are defined in Appendix A

2.3. Temperature distribution

In the presence of nanoparticles, we have to investigate the thermal characteristics of the fluid, and their effect on the flow of carbon nanotube particles suspended in water (SWCNTs). In this study the following models were used:

Maxwell's model [30],

$$\frac{k_{nf}}{k_f} = \frac{k_p + 2k_f - 2\phi(k_f - k_p)}{k_p + 2k_f + \phi(k_f - k_p)}. \tag{45}$$

Hamilton Crosser's (H-C) model [31],

$$\frac{k_{nf}}{k_f} = \frac{k_p + (n-1)k_f - (n-1)\phi(k_f - k_p)}{k_p + (n-1)k_f + \phi(k_f - k_p)}. \tag{46}$$

Xue's model [16],

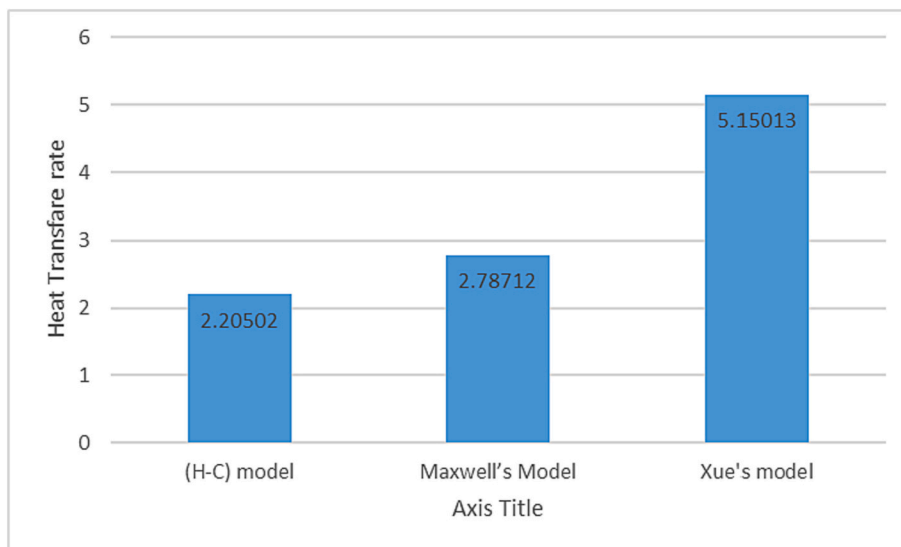


Fig. 14. Heat transfer rate at the wall for different volume fraction of SWCNTs.

$$\frac{k_{nf}}{k_f} = \left[\frac{1 - \phi + 2\phi \frac{k_{CNT}}{k_{CNT} - k_f} \ln\left(\frac{k_{CNT} + k_f}{2k_f}\right)}{1 - \phi + 2\phi \frac{k_f}{k_{CNT} - k_f} \ln\left(\frac{k_{CNT} + k_f}{2k_f}\right)} \right] \quad (47)$$

In the above equations, k_f denotes the base fluid’s thermal conductivity, k_{nf} denotes the nanofluids actual thermal conductivity, k_p denotes the thermal conductivity of nanoparticles, k_{CNT} denotes the thermal conductivity of SWCNTs, and ϕ denotes the nanotubes volume fraction.

Hence, the thermal transport characteristics [6] associated with this model and combined with electromagnetohydrodynamic flows through a corrugated microchannel is written as:

$$(\rho C_p)_{nf} \left[w^* \frac{\partial T^*}{\partial z^*} \right] = k_{nf} \left(\frac{\partial^2 T^*}{\partial x^{*2}} + \frac{\partial^2 T^*}{\partial y^{*2}} \right) + J + \psi^* \quad (48)$$

where T^* is the temperature of the fluid, ψ^* is the heat generation/absorption, J is the Joule heating,

$$J = (E_x^2 + E_z^2) \sigma_{nf}, \text{ and } k_{nf} \text{ is the thermal conductivity}$$

The classical non-dimensional temperature is now used for thermally fully evolved flow $T = \frac{k_f(T^* - T_w)}{q_w H}$ is now invariant of the axial coordinate for thermally fully developed flow, T_w is the channel wall temperature, which is constant across the cross-section of the channel, q_w is the average inward wall heat flow, of the channel. For thermally fully developed flow with constant heat flux, we may write:

$$\frac{\partial T^*}{\partial z^*} = \frac{\partial T_w^*}{\partial z^*} = \frac{\partial T_b^*}{\partial z^*} \quad (49)$$

where T_b^* is the bulk mean temperature. The overall energy balance on an element control volume can be expressed as

$$\frac{\partial T_b^*}{\partial z^*} = \frac{q_w}{H w_{av}^* (\rho C_p)_f} + \frac{J}{w_{av}^* (\rho C_p)_f} \quad (50)$$

Dimensionless variables are created using the non-dimensional variables in Eq. (50).

$$z = \frac{z^*}{H}, T = \frac{T^*}{T}, w_{0m} = \frac{w_{0m}^*}{u} \text{ and } \frac{k_{nf}}{k_f} = A_4$$

$$(\rho C_p)_{nf} = (\rho C_p)_f \left(1 - \phi + \phi \frac{(\rho C_p)_p}{(\rho C_p)_f} \right) = A_3 (\rho C_p)_f \quad (51)$$

$$\text{where, } j = \frac{(E_x^2 + E_z^2) \sigma_{nf} H}{q_w} \text{ and } \psi = \left(\frac{H \psi^*}{A_4 q_w} \right)$$

$$\therefore \frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} - \left(\frac{A_3}{A_4} \right) \left(\frac{w}{w_{0m}} \right) (1 + j) = -\frac{1}{A_4} j - \psi \quad (52)$$

with boundary conditions

$$T = 0 \text{ at } y_u = 1 + \varepsilon \sin(\lambda x) \quad (53)$$

$$T = 0 \text{ at } y_u = -1 \pm \varepsilon \sin(\lambda x) \quad (54)$$

By using the following transformation into the Eq. (52), we have

$$\theta = - \left(\frac{A_4}{A_3} \right) \frac{Ha^2 u_{av}}{1 + j} T + \frac{Ha^2}{A_2 (Ha^2 - \frac{A_1}{A_2} \alpha^2)} \Phi \quad (55)$$

We deduced that

$$\left(\frac{\partial^2 \theta}{\partial x^2} + \frac{\partial^2 \theta}{\partial y^2} \right) - \frac{A_1}{A_2} u = \left(\frac{1}{A_3} \right) \frac{Ha^2 u_{av}}{1 + j} j + \left(\frac{A_4}{A_3} \right) \frac{Ha^2 u_{av}}{1 + j} \psi \quad (56)$$

Defining the following variables

$$\gamma = \frac{A_1}{A_2}, \delta = \left(\frac{1}{A_3} \right) \frac{Ha^2 u_{av}}{1 + j} j + \left(\frac{A_4}{A_3} \right) \frac{Ha^2 u_{av}}{1 + j} \psi, \beta = \frac{Ha^2}{A_2 (Ha^2 - \frac{A_1}{A_2} \alpha^2)}$$

Then, the boundary conditions take the form

$$\theta = \beta \text{ at } y_u = 1 + \varepsilon \sin(\lambda x) \quad (57)$$

$$\theta = \beta \text{ at } y_u = -1 \pm \varepsilon \sin(\lambda x) \quad (58)$$

The Temperature function can be described by a regular perturbation expansion in small values of ε [25–27].

$$\theta(x, y) = \theta_0(x, y) + \varepsilon \theta_1(x, y) + \varepsilon^2 \theta_2(x, y) + \dots \quad (59)$$

A differential equation for powers of ε are obtained by substituting Eq. (59) into Eq. (56)

$$\varepsilon^0 : \frac{\partial^2 \theta_0}{\partial x^2} + \frac{\partial^2 \theta_0}{\partial y^2} - \gamma u_0 = \delta \quad (60)$$

$$\varepsilon^1 : \frac{\partial^2 \theta_1}{\partial x^2} + \frac{\partial^2 \theta_1}{\partial y^2} - \gamma u_1 = 0 \quad (61)$$

$$\varepsilon^2 : \frac{\partial^2 \theta_2}{\partial x^2} + \frac{\partial^2 \theta_2}{\partial y^2} - \gamma u_2 = 0 \quad (62)$$

In a Taylor series involving the mean wall positions $y = 1$ and $y = -1$, the boundary constraints (57) and (58), respectively, can be expanded as follows

$$\beta = \theta(x, 1) + \varepsilon \sin(\lambda x) \frac{\partial \theta}{\partial y}(x, 1) + \frac{\varepsilon^2}{2} \sin^2(\lambda x) \frac{\partial^2 \theta}{\partial y^2}(x, 1) + \dots \quad (63)$$

$$\beta = \theta(x, -1) \pm \varepsilon \sin(\lambda x) \frac{\partial \theta}{\partial y}(x, -1) + \frac{\varepsilon^2}{2} \sin^2(\lambda x) \frac{\partial^2 \theta}{\partial y^2}(x, -1) + \dots \quad (64)$$

We get for the corresponding boundary condition Eqs. (60), (61), and (62), respectively, by collecting terms of equal powers of ε .

$$\varepsilon^0 : \left\{ \theta_0(x, y) \Big|_{y=1} = \beta, \theta_0(x, y) \Big|_{y=-1} = \beta, \right. \quad (65)$$

$$\varepsilon^1 : \left\{ \begin{aligned} \theta_1(x, y) \Big|_{y=1} + \sin(\lambda x) \frac{\partial \theta_0(x, y)}{\partial y} \Big|_{y=1} &= 0, \\ \theta_1(x, y) \Big|_{y=-1} \pm \sin(\lambda x) \frac{\partial \theta_0(x, y)}{\partial y} \Big|_{y=-1} &= 0, \end{aligned} \right. \quad (66)$$

$$\varepsilon^2 : \left\{ \begin{aligned} \theta_2(x, y) \Big|_{y=1} + \sin(\lambda x) \frac{\partial \theta_1(x, y)}{\partial y} \Big|_{y=1} + \frac{\sin^2(\lambda x)}{2} \frac{\partial^2 \theta_0(x, y)}{\partial y^2} \Big|_{y=1} &= 0, \\ \theta_2(x, y) \Big|_{y=-1} \pm \sin(\lambda x) \frac{\partial \theta_1(x, y)}{\partial y} \Big|_{y=-1} + \frac{\sin^2(\lambda x)}{2} \frac{\partial^2 \theta_0(x, y)}{\partial y^2} \Big|_{y=-1} &= 0. \end{aligned} \right. \quad (67)$$

From Eqs. (60) and (65), we have

$$\theta_0^\pm(x, y) = \frac{e^{-Ay}}{2A^4(1 + e^{2A})} \left(A^2 (2(e^A - e^{Ay} - e^{A(2+y)} + e^{A+2Ay})) Q\gamma + A^2 e^{Ay} (1 + e^{2A}) (2\beta + (-1 + y^2)\delta) + (2e^A + 2e^{A+2Ay} + e^{Ay} (-2 + A^2 - A^2 y^2) + e^{A(2+y)} (-2 + A^2 - A^2 y^2)) \gamma B_1 \right) \quad (68)$$

From Eqs. (61) and (66), we have

$$\theta_1^+(x, y) = g_1^+(y) \text{Sin}[\lambda x], \quad (69)$$

$$\theta_1^-(x, y) = g_1^-(y) \text{Sin}[x\lambda]. \tag{70}$$

From Eqs. (62) and (67) then

$$\theta_2^+(x, y) = g_2^+(y) + f^+(y) \text{Cos}[2x\lambda], \tag{71}$$

$$\theta_2^-(x, y) = g_2^-(y) + f^-(y) \text{Cos}[2x\lambda], \tag{72}$$

where $g_1^+(y), g_1^-(y), g_2^+(y), g_2^-(y), f^+(y)$, and $f^-(y)$ are defined in Appendix A.

The Nusselt number, which represents the strength of convective heat exchange, can be calculated as follows

$$Nu^\pm = \frac{q^\pm(x) H}{k_f(T^* - T_w)}, \tag{73}$$

where

$$q^\pm(x) = -k_{nf} \left. \frac{\partial T^*}{\partial y^*} \right|_{y^* = y_w^*}. \tag{74}$$

Nusselt number can be written as

$$Nu^\pm = - \frac{k_{nf}}{k_f} \left. \frac{\partial T}{\partial y} \right|_{y=y_w}. \tag{75}$$

3. Discussion and outcomes

In this section, the behavior of the fluid supported by (SWCNTs) nanoparticles through corrugated walls is graphically explored. The problem was simulated by a set of partial differential equations and solved analytically using the perturbation method. The approximate analytical solutions were extracted using Mathematica software. The flow rate and heat transfer rate at different points in the microchannel were investigated.

To check the accuracy of the obtained results a comparison with the previously published data under the same conditions is shown in Tables 2 and 3, at $S = 0, \alpha = 0, \phi = 0, P_z = 0.5, \lambda = 0.25$, and $\varepsilon = 0.025$, and the values of non-dimensional parametric variables were calculated according to Table 4 [3,15,27].

It is worth mentioning that the channel's half height is considered to be $H \sim 40 \mu\text{m}$ for microfluid inquiry. The effects of the wavy roughness and porous of microchannel on electromagnetically driven flow and temperature were investigated using the following main parametric values [20,27].

$$O(\rho) \sim 1 \times 10^3 - 5.91 \times 10^3 \text{ kg m}^{-3},$$

$$O(\mu) \sim 0.001 \text{ kg ms}^{-1},$$

$$O(B_y) \sim 1 - 50\text{T},$$

$$O(E_x) \sim 0 - 10^4 \text{ V m}^{-1},$$

$$O(E_z) \sim 0 - 10^4 \text{ V m}^{-1},$$

The ranges of all physical variables used in this study are listed in Table 5

$$O(P1) \sim 10 \frac{\text{Pa}}{\text{m}},$$

$$O(S_j) \sim (-1) - (-15).$$

3.1. Effect of nanoparticles on the flow velocity

The effect of magnetic field (Hartmann number) on the volumetric flow rate shown in Table 3 for different values of nanoparticle concentration ϕ . Table 3 shows that the flow rate decreases as the Hartmann number increases under a concentration of nanoparticles owing to the

double effect of the body force caused by the applied electromagnetic forces. The first component is the ‘flow assist’ ($\sim \sigma_e E_x B_y$), which is formed between the electric and magnetic fields. The second component is the ‘opposite flux’ ($\sim \sigma_e B_y^2 u$), which is controlled by the Lorenz electromagnetic force, and such a force generates a normal component of the velocity while decreasing the main velocity of the flow. However, by investigating the effect of nanoparticle concentration on the flow rate, one can discover that as the nanoparticle concentration increases, the flow rate decreases due to increased fluid viscosity.

Figs. 2 and 3 depict the effect of the nanoparticle concentration on the velocity distribution in three dimensions and contour form for the case of the same phase walls. The velocity is affected only by the surface roughness, when the concentration of carbon particles (SWCNTs) is $\phi = 0.0$, as shown in Figs. 2a and 3a. When a small percentage of carbon particle concentration of $\phi = 0.05$ is added, the physical properties of the liquid were not greatly affected, as shown in Figs. 2b and 3b. and thus, the velocity was affected slightly. In contrast, when the percentage of concentration increased to $\phi = 0.1$ or $\phi = 0.2$ and with the surface roughness, the velocity decreased insignificantly, as shown in Figs. 2c, d, 3c, and d. Fig. 4a and b show the contour plot for the velocity distribution for different values of the nanoparticle concentration at $S = 50$, and $Ha = 1$. When the electric field increases, the velocity increases, and the thickness of the boundary layer increases. Figs. 5 and 6 show the effect of nanoparticle concentration when the phase difference between the two walls is 180° . Figs. 5a and 6a show that the velocity is affected only by the surface roughness, when the concentration of carbon particles (SWCNTs) is $\phi = 0.0$, but the undulating phenomenon occurs because of the phase difference. When a small percentage of carbon particle concentration of $\phi = 0.05$ is added, the speed decreases, but the wavy phenomenon increased due to the phase difference, as shown in Figs. 5b and 6b. When the percentage of concentration increased to $\phi = 0.1$ or $\phi = 0.2$ and with the surface roughness, the velocity decreased significantly, as shown in Figs. 5c, 6c, and d.

By increasing the electric and the magnetic fields to $S = 50, Ha = 1$, as shown in Fig. 7, the flow increases in the middle region and decreases as we approach the wall's region, with an increase in velocity also occurring when particles are present. Layers of liquids are greatly affected by the Lorenz force, which increases when the Hartmann number (Ha) increases and in the presence of a tangential electric field S . The impact of this force is observed and is clearly shown in the previous figures. Fig. 8 depicts the variance of the velocity distribution at $S = 50, Ha = 0.5$, in both cases of corrugated/flat channel; one can observe, a reduction in the velocity at the center of the channel occurs due to the corrugation of the walls and reduces more in the presence of the nanoparticles. Physically, the presence of nanoparticles raises the fluid viscosity which reflected in the velocity by the reduction.

Fig. 9 shows the average velocity ω_m as the Hartman number Ha increases if the phase difference between the two walls is 0° (represented by the solid lines) and 180° (represented by the dashed lines). The mean velocity ω_m gradually reaches a maximum value as the Hartman number increases from 1.0 to 1.5 and then decreases reduces to a minimum value. Furthermore, the presence of nanoparticles reduces the mean velocity of the flow, particularly in the opposite phase channel.

3.2. Effect of wall roughness on flow temperature in the presence of nanofluidic particles

After deriving the heat Eq. (56) and using Maxwell's model with n as the parameter, the structural coordination of nanoparticles is given by $3/\xi$, where ξ is the spherically of nanoparticles. For spherical nanoparticles, $\xi = 1$ or $n = 3$, but the (H-C) model reduces nanoparticles to cylindrical shapes, with $\xi = 0.5$ or $n = 6$. The goal of the temperature study is to examine the effect of the different models of heat conduction mentioned. Fig. 10 shows the behavior of both the transverse magnetic and electric fields. It helps to regulate the temperature of the flowing

liquid. The thermal distribution shows that as the number of nanoparticles increases, the temperature increases. When the transverse magnetic and electric fields are increased $S = 25, Ha = 0.5$, the degree increases when the nanoparticles are increased, as shown in Fig. 11. When studying the effect of nanoparticles on heat transfer, we observed that the rate of the increase of the Nusselt number Nu increases as the concentration of nanoparticles increases. Fig. 12 shows the Nusselt number with increasing Nu of Hartmann number Ha . If the phase difference between the two walls is 0° (represented by the solid lines) and 180° (represented by dashed line), the electric and magnetic fields control the rate of heat transfer. It is observed that from the energy equation, the joule temperature increases as the transverse magnetic and electric fields increase.

Therefore, the temperature spread is further increased by increasing the transverse magnetic and electric fields. Fig. 13 shows the effect of nanoparticles on heat generation/absorption ψ , when the phase difference between the two walls is 0° (represented by the solid lines) and 180° (represented by dashed lines), the rate of increase of the Nusselt number Nu increases when the concentration of nanoparticles increases.

Table 5 shows the numerical values of heat transfer rates for changes in the nanoparticle concentration. A comparison of the Maxwell model, the H-C model, and the Xue model is made in this table. The H-C model has the lowest heat transfer rate by 5.19% at 0.2 concentration. The heat transfer rate of the Maxwell model is higher than that of the H-C model by 15.01% at 0.2 concentrations. The Xue model increases the heat transfer rate by 23.33% at 0.2 concentrations. We also note that all three models, increasing the fracture size in the nanotubes increases the heat transfer rate. When we compare the Xue model with the other models, we note that the heat transfer rate predicted by the Xue model is higher than that by the other models. These results agree with previous findings by [11]. Fig. 14 presents a comparison between the three models of thermal characteristics suggested in this study, it is clear that the rate of heat transfer for Xue's model is the highest one by 84.78 % compared with Maxwell's model and by 133.56 % compared with the H-C model.

4. Conclusion

The flow and heat rate effects of SWCNTs, as well as wall surface roughness, were studied through a fine corrugated channel in the presence of magnetic and electric fields at a constant pressure gradient, and the following conclusions were reached.

- Due to the phase difference between the two walls, the speed and temperature distribution depend on the shape of the channel wall.
- When the corrugation increases, the phenomenon of ripple speed and temperature distribution becomes apparent.
- One can observe that the velocity distribution decreases as the concentration increases due to an increase in fluid viscosity.
- The impact of the transverse electric field (S) on the velocity contour is visible. It is observed that increasing the transverse electric field

improves the fluid velocity and makes the wavy phenomenon of the flow is more visible.

- The addition of SWCNTs, reduces the fluid velocity at the center of the channel by providing resistance to the fluid motion.
- When the transverse electric field is increased, there is a spread and temperature rise.
- The concentration of SWCNTs influence enhances the rate of heat transfer.
- The rate of increase of the Nusselt number Nu increases when the concentration of nanoparticles increases.
- Heat generation/absorption increases as the transverse magnetic and electric fields increase with the change in the concentration of SWCNTs.
- Xue's model has the highest heat transfer rate compared to the Maxwell and H-C models.

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CRediT authorship contribution statement

Ahmed Y. Sayed: Conceptualization, Validation, Writing - Reviewing and Editing.

Shaaban I. Ahmed: Data curation, Writing - Original draft preparation and Methodology

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Mohamed S. Abdel-wahed: Methodology, Software, Validation and Reviewing

Declaration of competing interest

The authors declare that they have no competing interests.

Data availability

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Appendix A

$$S_1^2 = \alpha^2 + \lambda^2 \tag{A-1}$$

$$S_2^2 = \alpha^2 + 4\lambda^2 \tag{A-2}$$

$$B_1 = Ha^2 \frac{A_2}{A_1^2} P_z + \frac{A_2^2}{A_1^2} Ha^3 S \tag{A-3}$$

$$B_2^2 = \lambda^2 + A^2 \tag{A-4}$$

$$B_3^2 = 4\lambda^2 + A^2 \tag{A-5}$$

$$B_4 = \frac{(A^2Q + B_1)Tanh[A]}{A} \tag{A-6}$$

$$g_1^+(y) = \frac{Csch[2\lambda]Sin[x\lambda]}{\lambda^2 - B_2^2} (2\gamma Cosh[\lambda](Sinh[y\lambda] - Csch[B_2]Sinh[\lambda]Sinh[yB_2])B_4 - (\lambda^2 - B_2^2)(Sinh[(1+y)\lambda]B_5 + Sinh[\lambda - y\lambda]B_6)) \tag{A-7}$$

$$g_1^-(y) = \frac{1}{(-1 + e^{4\lambda})(\lambda^2 - B_2^2)} e^{-y\lambda} Sin[x\lambda] ((-1 + e^{2\lambda})\gamma(-e^\lambda(1 + e^{2y\lambda}) + e^{y\lambda}(1 + e^{2\lambda})Cosh[yB_2]Sech[B_2])B_4 - 2e^{(2+y)\lambda}(\lambda^2 - B_2^2)(Sinh[(1+y)\lambda]B_5 - Sinh[\lambda - y\lambda]B_6)) \tag{A-8}$$

$$g_2^+(y) = \frac{(1+y)B_7 - (-1+y)B_8}{4} + \frac{\gamma(A^2Q(1 - Cosh[Ay]Sech[A]) + (1 - Cosh[Ay]Sech[A])(B_1 - 2Coth[B_2]B_2B_4))}{4A^2} \tag{A-9}$$

$$g_2^-(y) = \frac{(1+y)B_9 - (-1+y)B_{10}}{4} + \frac{\gamma(A^2Q(1 - Cosh[Ay]Sech[A]) + (1 - Cosh[Ay]Sech[A])(B_1 - 2B_2B_4Tanh[B_2]))}{4A^2} \tag{A-10}$$

$$f^+(y) = \frac{1}{4(-1 + e^{8\lambda})(4\lambda^2 - B_3^2)} (A^2Q\gamma(2e^{2\lambda}(-1 + e^{4\lambda})Cosh[2y\lambda] - (-1 + e^{8\lambda})Cosh[yB_3]Sech[B_3]) + 2e^{4\lambda}(\gamma(2Cosh[2y\lambda]Sinh[2\lambda] - Cosh[yB_3]Sech[B_3]Sinh[4\lambda])(B_1 - 2Coth[B_2]B_2B_4) - 2(4\lambda^2 - B_3^2)(Sinh[2(1+y)\lambda]B_7 + Sinh[2\lambda - 2y\lambda]B_8))) \tag{A-11}$$

$$f^-(y) = \frac{1}{16\lambda^2 - 4B_3^2} (A^2Q\gamma(Cosh[2y\lambda]Sech[2\lambda] - Cosh[yB_3]Sech[B_3]) - 2Csch[4\lambda](4\lambda^2 - B_3^2)(Sinh[2(1+y)\lambda]B_9 + Sinh[2\lambda - 2y\lambda]B_{10}) + \gamma(Cosh[2y\lambda]Sech[2\lambda] - Cosh[yB_3]Sech[B_3])(B_1 - 2B_2B_4Tanh[B_2])) \tag{A-12}$$

$$B_5 = \frac{\gamma B_1(-A + Tanh[A]) + A^2(A\delta + Q\gamma Tanh[A])}{A^3} \tag{A-13}$$

$$B_6 = \frac{\gamma B_1(A - Tanh[A]) - A^2(A\delta + Q\gamma Tanh[A])}{A^3} \tag{A-14}$$

$$B_7 = \frac{1}{2}(-Q\gamma - \delta) - \frac{Csch[2\lambda](2\gamma Cosh[\lambda](\lambda Cosh[\lambda] - Coth[B_2]Sinh[\lambda]B_2)B_4 - \lambda(\lambda^2 - B_2^2)(Cosh[2\lambda]B_5 - B_6))}{\lambda^2 - B_2^2} \tag{A-15}$$

$$B_8 = \frac{1}{2}(-Q\gamma - \delta) - \frac{\gamma(\lambda Coth[\lambda] - Coth[B_2]B_2)B_4 + \lambda Csch[2\lambda](\lambda - B_2)(\lambda + B_2)(-B_5 + Cosh[2\lambda]B_6)}{\lambda^2 - B_2^2} \tag{A-16}$$

$$B_9 = \frac{1}{2}(-Q\gamma - \delta) - \frac{-2e^{2\lambda}\lambda(\lambda^2 - B_2^2)(Cosh[2\lambda]B_5 + B_6) + \gamma B_4((-1 + e^{2\lambda})^2\lambda + (-1 + e^{4\lambda})B_2Tanh[B_2])}{(-1 + e^{4\lambda})(\lambda^2 - B_2^2)} \tag{A-17}$$

$$B_{10} = \frac{1}{2}(-Q\gamma - \delta) + \frac{-2e^{2\lambda}\lambda(\lambda^2 - B_2^2)(B_5 + Cosh[2\lambda]B_6) + \gamma B_4((-1 + e^{2\lambda})^2\lambda - (-1 + e^{4\lambda})B_2Tanh[B_2])}{(-1 + e^{4\lambda})(\lambda^2 - B_2^2)} \tag{A-18}$$

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